



Shigley's mechanical engineering design 10th edition solutions chapter 8

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Fresno, California 93725. United States. Custom manufacturer of springs for trucking, automotive, mining, military, agricultural, and construction industries. Types of springs include coil, leaf, hot and cold wound, precision, single and double torsion, conical, music wire, stainless steel, pigtail, compression and extension springs. -Anderson Quality Spring Manufacturing, Inc. Canby, Oregon 97013. United States. Custom manufacturer of coil springs include compression, extension, suspension, spiral torsion, and wave springs. Capabilities include forming, machining, milling, tapping, threading, welding and drilling services. -Duer Carolina Coil, Inc. Reidville, South Carolina 29375-0730. United States. Manufacturer of precision cold & hot wound mechanical compression, extension & torsion springs. From .250 in. wire to 3.125 in. bar in virtually any material. Engineering assistance available for original design or redesign of spring requirements. Quality control from material procurement, throughout the production process, to packaging at time of shipment. -Alco Spring Industries, Inc. Chicago Heights, Illinois 60411. United States. Manufacturer Of Precision Hot Wound Coil Springs From 3/8ths To 4 Inches In Bar Diameter With An Outside Diameter Range From 1 1/2 Inches To 24 Inches & A Free Height Maximum Of 66 Inches. Engineering & Design Assistance Is Available To Meet Specific Customer Requirements. Quality Control Standards For The Entire Manufacturing Process To Meet Specific Customer Requirements. Company. Garland, Texas 75042. United States. Custom manufacturer of precision wire forms and springs including compression, extension, and torsion. Specialized in a wide variety of complex spring-tempered wire forms. Capabilities include automatic CNC machining, hand winding or hand forming, hand operated or automatic grinding, stress relief heating, and painting. Special packaging is available upon request. Page 2 You need to log in to continue DenunciarChapter 8 Note to the Instructor for Probs. 8-41 to 8-44. These problems, as well as many others in this chapter are best implemented using a spreadsheet. 8-1 (a) Thread depth= 2.5 mm Ans. Width = 2.5 mm Ans. dm = 25 - 1.25 -1.25 = 22.5 mm dr = 25 - 5 = 20 mm l = p = 5 mm Ans. (b) Thread depth = 2.5 mm Ans. Width at pitch line = 2.5 mm Ans. dm = 22.5 mm dr = 20 mm l = p = 5 mm Ans. 8-2 From Table 8-1, 1.226 869 0.649 519 1.226 869 0.649 519 0.938 194 2 r m d d p d d p d p d p d d p 2 2(0.938 194) . 4 4t dA d p Ans 8-3 From Eq. (c) of Sec. 8-2, tan 1 tan tan 2 2 1 tan R R m m R fP F f P d F d fT f 0 / (2) 1 tan 1 tantan . / 2 tan tan R m T F l f fe A T F d f f ns Chap. 8 Solutions - Rev. A, Page 1/69 Using f = 0.08, form a table and plot the efficiency curve. , deg. e 0 0 0 0.678 20 0.796 30 0.838 40 0.8517 45 0.8519 8-4 Given F = 5 kN, l = 5 mm, and dm = d p/2 = 25 5/2 = 22.5 mm, the torgue required to raise the load is found using Eqs. (8-1) and (8-6) 5 22.5 5 0.09 22.5 5 0.06 45 15.85 N m . 2 22.5 0.09 5 22.5 0.09 22.5 5 5 0.06 45 7.83 N m . 2 22.5 0.09 5 2L T A 5 2R T A ns The torque required to lower the load, from Eqs. (8-2) and (8-6) is ns Since TL is positive, the thread is self-locking. From Eq.(8-4) the efficiency is 550.251.215.85 e Ans 8-5 Collar (thrust) bearings, at the bottom of the screws, must be in compression. Whereas, tension specimens and their grips must be in tension. Both screws must be of the same-hand threads. 8-6 Screws rotate at an angular rate of 1720 28.67 rev/min 60 n Chap. 8 Solutions - Rev. A, Page 2/69 (a) The lead is 0.25 in, so the linear speed of the press head is V = 28.67(0.25) = 7.17 in/min Ans. (b) F = 2500 lbf/screw o 2 0.25 / 2 1.875 in sec 1 / cos(29 / 2) 1.033 md Eq. (8-5): 2500(1.875) 0.25 (0.05)(1.875)(1.033) 221.0 lbf · in 2 (1.875) 0.05(0.25)(1.033)R T Eq. (8-6): 2500(0.08)(3.5 / 2) 350 lbf · in 350 221.0 571 lbf · in/screw 571(2) 20.04 lbf · in 60(0.95) 20.04(1720) 0.547 hp . 63 025 63 025 c total motor T T T T H A 8-7 Note to the Instructor: The statement for this problem in the first printing of this edition was ns vague regarding the effective handle length. For the printings to follow the statement "The overall length is 4.25 in." will be replaced by "A force will be applied to the handle at a radius of 123 in from the screw centerline." We apologize if this has caused any inconvenience. 3 3 3.5 in 3.5 3 33.5 3.125 8 8 41 kpsi 32 32(3.125) 41 000 (0.1875) 8.49 lbf F ns 3.5(8.49) 29.7 lbf · in TA (b) Eq. (8-5), 2 = 60, l = 1/10 = 0.1 in, f = 0.15, sec = 1.155, p = 0.1 in Chap. 8 Solutions - Rev. A, Page 3/69 clamp clamp clamp 3 0.649 519 0.1 (0.15)(0.6850)(1.155) 2 (0.6850) 0.15(0.1)(1.155) 0.075 86 29.7 392 lbf . 0.075 86 0.075 86 m R R R d y y L T F M L F F S M FS d F ns (c) The column has one end fixed and the other end pivoted. Base the decision on the mean diameter column. Input: C = 1.2, D = 0.685 in, A = (0.6852)/4 = 0.369 in 2, Sy = 41 kpsi, E = 30(106) psi, L = 6 in, k = D/4 = 0.171 25 in, L/k = 35.04. From Eq. (4-45), 1/21/2 2 62 1 2 1.2 30 102 131.7 41 000y l CE k S FTTFA (d) This is a subject for class discussion. From Eq. (4-46), the limiting clamping force for buckling is 2 clamp cr 23 3 3 6 1 2 41 10 10.369 41 10 35.04 14.6 10 lbf 2 1.2 30 10 y y S lF P A S k CE Ans 8-8 T = 8(3.5) = 28 lbf in 3 1 0.6667 in 4 12m d l = 1 6= 0.1667 in, = 0292 = 14.50, sec 14.50 = 1.033 From Eqs. (8-5) and (8-6) total 0.1667 0.15 0.6667 1.033 0.15 10.6667 0.1542 2 0.6667 0.15 0.1667 1.033 2 FFT F 28 182 lbf . 0.1542 F Ans Chap. 8 Solutions - Rev. A, Page 4/69 8-9 dm = 1.5 0.25/2 = 1.375 in, l = 2(0.25) = 0.5 in From Eq. (8-1) and Eq. (8-6) 3 32.2 10 (1.375) 2.2 10 (0.15)(2.25)0.5 (0.10)(1.375) 2 (1.375) 0.10(0.5) 2 330 371 701 lbf · in RT Since $n = V/l = 2/0.5 = 4 \text{ rev/s} = 240 \text{ rev/min so the power is } 701 240 2.67 \text{ hp} \cdot 63 025 63 025 \text{ TnH A} \text{ ns}$ 8-10 dm = 40 48-11 (a) Table A-31, = 36 mm, l = p = 8 mm From Eqs. (8-1) and (8-6) 36 8 (0.14)(36) 0.09(100) 2 (36) 0.14(8) 2 (3.831 4.5) 8.33 N · m (in kN) 2 2 (1) 2 rad/s 3000 477 N · m 2 477 57.3 kN . 8.33 F FT F F F n H T HT F Ans 57.3(8) 0.153 . 2 2 (477) Fle A T ns nut height H = 12.8 mm. L \geq l + H = 2(15) + 12.8 = 42.8 mm. Rounding up, L = 45 mm Ans. (b) From Eq. (8-14), LT = 2d + 6 = 2(14) + 6 = 34 mm From Table 8-7, ld = L LT = 45 34 = 11 mm, lt = 1 ld = 2(15) 11 = 19 mm, Ad = (142) / 4 = 153.9 mm2. From Table 8-7, ld = L LT = 45 34 = 11 mm, lt = 1 ld = 2(15) 11 = 19 mm, Ad = (142) / 4 = 153.9 mm2. From Table 8-7, ld = L LT = 45 34 = 11 mm, lt = 1 ld = 2(15) 11 = 19 mm, Ad = (142) / 4 = 153.9 mm2. From Table 8-7, ld = L LT = 45 34 = 11 mm, lt = 1 ld = 2(15) 11 = 19 mm, Ad = (142) / 4 = 153.9 mm2. From Table 8-7, ld = L LT = 45 34 = 11 mm, lt = 1 ld = 2(15) 11 = 19 mm, Ad = (142) / 4 = 153.9 mm2. From Table 8-7, ld = L LT = 45 34 = 11 mm, lt = 1 ld = 2(15) 11 = 19 mm, Ad = (142) / 4 = 153.9 mm2. From Table 8-7, ld = L LT = 45 34 = 11 mm, lt = 1 ld = 2(15) 11 = 19 mm, Ad = (142) / 4 = 153.9 mm2. From Table 8-7, ld = L LT = 45 34 = 11 mm, lt = 1 ld = 2(15) 11 = 19 mm, Ad = (142) / 4 = 153.9 mm2. From Table 8-7, ld = L LT = 45 34 mm From Table 8-7, ld = L LT = 45 34 mm. $115\ 207\ 874.6\ MN/m$. $153.9\ 19\ 115\ 11\ d\ t\ b\ d\ t\ t\ d\ A\ A\ Ek\ A\ A\ I\ A\ I$ ns (c) From Eq. (8-22), with $I = 2(15) = 30\ mm$ $0.5774\ 207\ 140.5774\ 3\ 116.5\ MN/m$. $0.5774\ 30\ 0.5\ 142\ln\ 5\ 2\ln\ 50.5774\ 2.5\ 0.5774\ 30\ 2.5\ 14\ mk\ Ed\ Ans\ I\ d\ I\ d$ 8-12 (a) Table A-31, nut height H = 12.8 mm. Table A-33, washer thickness t = 3.5 mm. Thus, the grip is l = 2(15) + 3.5 = 33.5 mm. $L \ge l + H = 33.5 + 12.8 = 46.3 \text{ mm}$. Rounding up L = 50 mm Ans. (b) From Eq. (8-14), LT = 2d + 6 = 2(14) + 6 = 34 mm From Table 8-7, ld = L LT = 50 34 = 16 mm, lt = l ld = 33.5 16 = 17.5 mm, $Ad = (142) / 4 = 153.9 \text{ mm}^2$. From Table 8-1, $At = 115 \text{ mm}^2$. From Eq. (8-17) 153.9 115 207 808.2 MN/m . 153.9 115 207 808.2 MN/m . 153.9 115 207 808.2 MN/m . 153.9 II = 20 \text{ mm}^2. 17.5 You're Reading a Free Preview Pages 7 to 8 are not shown in this preview. You're Reading a Free Preview Pages 13 to 26 are not shown in this preview. The Solution Manual for Shigleys Mechanical Engineering Design 10th Edition by Budynas (Chapters 18 and 19 not included) is THE solution to your homework problems. Download your FREE Sample NOW! Catch up on homework in no time with this indispensable solution manual. It's filled with answers to questions at the end of chapters, problems, case studies and more directly from your text book. 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